Towards Large Eddy Simulation for Turbo-machinery Flows

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In Memory of Prof. Jaw-Yen Yang











Outline

Introduction to large eddy simulations (LES)

- Key pacing items enabling LES with high-order adaptive methods
 - High-order methods
 - High-order mesh generation
 - SGS models
- Sample demonstrations
- Conclusions





Introduction

Approaches to compute turbulent flows

- RANS: model all scales
- LES: resolve large scales while modeling small scales
- DNS: resolve all scales
- ➤What is LES
 - $\hfill \ensuremath{\,^\circ}$ Partition all scales into large scales and small sub-grid scales with a low pass filter with width Δ
 - Solve the filtered Navier-Stokes equations with a SGS closure model
 - A compromise between RANS and DNS





RANS Inadequate for Many Applications











LES – the Challenges

- \succ How to choose the filter width Δ
- > How to resolve the disparate length and time scales in the turbulent flow field
- > How to handle complex geometries
- > How to resolve very small turbulence scales in the boundary layer
- Discontinuity capturing
- Parallel performance on extreme scale computers
- Post-processing and visualization of large data sets





Key Pacing Items in LES

- >High-order methods capable of handling unstructured meshes to deal with complex geometry
- High-order meshes resolving the geometry and viscous boundary layers
 - Coarse meshes (because internal degrees of freedom are added)
- >Quality of SGS models
- Wall models to decrease the number of cells in the boundary layer



High order methods





High-Order CFD Methods Needed

>All of the challenges demand more accurate, efficient and scalable design tools in CFD $Error \propto h^{p>2}$

4th order

- Better engine simulation tools
- Better design tools for high-lift configurations







Popular High-Order Methods

Compact difference method >Optimized difference method >ENO/WENO methods >MUSCL, PPM and K-exact FV > Residual distribution methods Discontinuous Galerkin (DG) >Spectral volume (SV)/spectral difference (SD) Flux reconstruction/Correction procedure via reconstruction

Unstructured grid

≻…





How to Achieve High-Order Accuracy

- Extend reconstruction stencil
 - Finite difference, compact
 - Finite volume, ENO/WENO, ...
- > Add more internal degrees of freedom
 - Finite element/spectral element, discontinuous Galerkin
 - Spectral volume (SV)/spectral difference (SD), flux reconstruction (FR) or correction procedure via reconstruction (CPR), ...
- > Hybrid approaches
 - PnPm, rDG, hybrid DG/FV, ...





Extending Stencil vs. More Internal DOFs

- Simple formulation and easy to understand for structured mesh
- Complicated boundary conditions: high-order one-sided difference on uniform grids may be unstable
- Not compact

- Boundary conditions trivial with uniform accuracy
- Non-uniform and unstructured grids
 - Reconstruction universal
- >Scalable
 - Communication through immediate neighbor





Review of the Godunov FV Method

Consider

$$\frac{\partial u}{\partial t} + \frac{\partial f(u)}{\partial x} = 0$$

$$V_{i}$$

$$i-1/2$$

$$i+1/2$$

Integrate in V_i

$$\int_{V_i} \left(\frac{\partial u}{\partial t} + \frac{\partial f}{\partial x} \right) dx = \frac{\partial \overline{u}_i}{\partial t} \Delta x_i + \int_{i-1/2}^{i+1/2} \frac{\partial f}{\partial x} dx$$
$$= \frac{\partial \overline{u}_i}{\partial t} \Delta x_i + (f_{i+1/2} - f_{i-1/2}) = 0$$

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FR/CPR

Developed by Huynh in 2007 and extended to simplex by Wang & Gao in 2009, ...

> It is a differential formulation like "finite difference"

$$\frac{\partial U_i(x)}{\partial t} + \frac{\partial F_i(x)}{\partial x} = 0, \quad U_i(x) \in P^k, \quad F_i(x) \in P^{k+1}$$

>The DOFs are solutions at a set of "solution points"





CPR (cont.)

> Find a flux polynomial $F_i(x)$ one degree higher than the solution, which minimizes

$$\left\|\tilde{F}_{i}(x) - F_{i}(x)\right\|$$

>The use the following to update the DOFs





CPR – DG

> If the following equations are satisfied

$$\int_{V_i} \left[\tilde{F}_i(x) - F_i(x) \right] dx = 0$$
$$\int_{V_i} \left[\tilde{F}_i(x) - F_i(x) \right] x dx = 0$$

>The scheme is DG!







High order mesh generation



The Need for Coarse, High-Order Meshes

- Internal degrees of freedom are added such that meshes with ~100,000 elements may be sufficient to achieve engineering accuracy
- If boundaries are still represented by linear facets, large errors are generated









Low versus High-Order Meshes: An Example in 2D



low-order



high-order







CAD Free, Low to High-Order Mesh Conversion

(released free of charge, just google meshCurve)





The Mission





Main Features

- > CGNS meshes : 3D, unstructured, multi-zone, multi-patch
- >CAD-free operation
- Feature-curve preservation
- >Easy-to-use, cross-platform graphical user interface interface
- Interactive 3D graphics
- Solid code base with minimal reliance on outside software libraries.
- Reasonably low memory footprint and fast operation on a desktop computer.
- >Available for: Linux, MS Windows and Mac platforms





Demo Video









SGS Models with the Burgers' Equation





Our Venture into LES

Solve the filtered LES equations using

- FR/CPR scheme
- 3 stage SSP Runge-Kutta scheme for time marching
- Implemented 3 SGS models
 - Static Smagorinsky (SS) model
 - Dynamic Smagorinsky (DS) model
 - ILES (no model)
- Attempted several benchmark problems
 - Flow over a Cylinder (ILES)
 - Isotropic turbulence decay (SS, DS, ILES)
 - Channel flow (SS, DS, ILES)





LES Results – Isotropic Turbulence Decay



Why ILES Performs Better Consistently

- >No good explanation!
- So we decided to evaluate SGS models using the 1D Burgers' equation
 - High resolution DNS can be easily carried out
 - True stress can be computed based on DNS data
 - Both a priori and a posteriori studies can be performed
 - Yes, the physics of 1D Burger's equation is vastly simpler than the Navier-Stokes equations, but if a SGS model has any chance for 3D Navier-Stokes equations, it must perform well for the 1D Burger's equation

Filtered Burgers' Equation

1D Burgers' equation

$$\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} = v \frac{\partial^2 u}{\partial x^2}$$

Filter the equation with a box filter

$$\frac{\partial \hat{u}}{\partial t} + \hat{u}\frac{\partial \hat{u}}{\partial x} = v\frac{\partial^2 \hat{u}}{\partial x^2} - \frac{\partial \tau}{\partial x}.$$

where

$$\boldsymbol{\tau} = \frac{1}{2}\hat{u}\hat{u} - \frac{1}{2}\hat{u}\hat{u}.$$

SGS Models Evaluated

Static Smagorinsky model (SS)
Dynamic Smagorinsky model (DS)
Scale similarity model (SSM)
Mixed model (MM) of SSM and DS
Linear unified RANS-LES model (LUM)
ILES (no model)

Numerical Method and Problem Setup

- Numerical method
 - 3rd order FR/CPR scheme
 - Viscous flux is discretized with BR2
 - Explicit SSP 3 stage Runge-Kutta scheme
- Problem setup
 - Domain [-1, 1] with periodic boundary condition
 - The initial solution contains 1,280 Fourier modes satisfying a prescribed energy spectrum with random phases
 - The DNS needs 2,560 cells to resolve all the scales
 - The filter width: $\Delta = 32 \Delta x_{DNS}$
 - Various mesh resolutions for LES $\Delta x_{LES}/\Delta = 1$, 1/2, 1/4, 1/8

Initial Condition

The DNS Results

Comparison of SGS Stresses (A Priori)

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Lessons Learned about SGS Models

- In both a priori and a posteriori tests with the 1D Burgers' equation
 - SGS stresses generated by static, dynamic Smagorisky and LUM models show no correlation with the true stress
 - SSM (and Mixed model) consistently produces stresses with the best correlation with the true stresses
- When the modeling error is dominant, SSM and MM perform the best. When the truncation error is dominant, no model shows any advantage. ILES is preferred
- > For methods with dissipation, DO NOT use SGS models. For almost all LES simulations, truncation errors are dominant ($\Delta = h$), the best choice is ILES.

Example Applications

Parallel Efficiency: Strong Scalability Test

- > Compare packing/unpacking vs direct data exchange
- > P3 100 RK3 iterations on BlueWater; 125,000 Hex elements
- > 3D inviscid vortex propagation: 72% at 8192 cores (15 elements/core)
- > 3D viscous Couette Flow: 68% at 16384 cores (8 elements/core)

Periodic Hill

- Benchmark problem adopted by the international workshops for high-order CFD methods
- Re = 2,800 and 10,595
- Accurate prediction of separation and reattachment points is a key challenge
- P3 FR/CPR+3rd order SSP Runge-Kutta

Periodic Hill

Iso-surface of Q colored by streamwise velocity at $Re_b=10595$ (hybrid)

Periodic Hill (Re = 2,900)

Mean streamline

Periodic Hill (Re = 10,595)

Mean streamline

Separation and Reattachment Points

Velocity Profiles, **Re = 2,800**

Velocity Profiles, **Re = 10,595**

Uncooled VKI Vane Case - Benchmark

Reynolds number: 584,000, Mach exit: 0.94
 No. of hexahedral elements: 511,744
 nDOFs/equ at p5 (6th order): 110.5M

- Boundary conditions
 - Inlet: fix total p and total T and flow angle
 - Wall: no split and iso-thermal
 - Exit: fix p
 - Periodic on the rest
- Some challenges
 - There are supersonic regions and shock waves
 - Heat transfer is difficult to predict

Simulation Process

- Start the simulation from p0 (1st order), and then restart at higher orders. This is much more robust than directly starting at high order
- Monitor the CI and Cd histories on the main blades to determine the start time for averaging
- > P-refinement studies used to assess the accuracy and mesh and order independence

Q-Criterion and Computational Schlierens

46

Computational Schlierens

C50/0[0 C12.6 20 12600 12600 C36 3/2 (5)/0[9 12620 C2/0 0 0 C12.6 20.00

FDL3DI – sixth order compact scheme

FR/CPR - sixth order

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Comparison of Heat Transfer

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Summaries

>Outlined the challenges in LES

Focused on several pacing items for LES

- High order methods
- High-order mesh generation
- SGS models

Presented several demonstration cases to show the capability

Future work includes better wall models and efficient time integration schemes for extreme scale computers

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